Effect of charged line defects on conductivity in graphene: Numerical Kubo and analytical Boltzmann approaches

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SCREEN PRINTED PZT/PZT THICK FILM BIMORPH MEMS CANTILEVER DEVICE FOR VIBRATION ENERGY HARVESTING

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ABSTRACT

We present a MEMS-based PZT/PZT thick film bimorph vibration energy harvester with an integrated silicon proof mass. The most common piezoelectric energy harvesting devices utilize a cantilever beam of a non piezoelectric material as support beneath or in-between the piezoelectric material. It provides mechanical support but it also reduces the power output. Our device replaces the support with another layer of the piezoelectric material, and with the absence of an inactive mechanical support all of the stresses induced by the vibrations will be harvested by the active piezoelectric elements.

KEYWORDS

Energy harvester, MEMS, thick film, PZT, bimorph

INTRODUCTION

With the recent development in low power electronics and wireless systems it has become increasingly interesting to use energy harvesters that harness ambient energies to empower electronic sensor systems instead of using electrochemical batteries. The limited lifespan of traditional electrochemical batteries requires periodical replacement and the batteries often take up significant volume in the total sensing system. The MEMS energy harvester addresses these two issues by having large energy to volume ratio [1] and by scavenging energies from external sources, thus the lifetime is only limited by material decay or fatigue. This will reduce the frequency of periodic maintenance from months to years, and maybe even decades.

The most common ambient power sources are solar, thermal, mechanical or unused RF energy. Harvesting vibration energy from a mechanical noise source can either be based on electrostatic, electromagnetic or piezoelectric conversion [2]. The interest in harvesting energy from ambient mechanical vibrations has resulted in a number of review articles in recent years [1-4]. In particular the piezoelectric transduction method has received significant attention. A typical piezoelectric energy harvester is based on a bimorph cantilever beam, which consists of the active piezoelectric ceramic with metal electrodes on both side and a passive mechanical support structure, anchored at one end and with a proof mass at the other [5-8].

There are many different piezoelectric materials available, however the materials of most interest for application in MEMS devices are PZT (Pb(ZrxTi1-x)O3), zinc oxide (ZnO) [9], [10] and aluminium nitride (AlN) [11], [12]. Among these, PZT is the most commonly used in energy harvesters.

Energy harvesters with bulk PZT have been presented in [13] and [14]. These energy harvesters have large dimensions and are thus not compatible with small sensor systems. Energy harvesters made with thin film deposition methods, such as sputtering [15] or sol-gel spin-on [16], [17] are characterized by a very thin layer PZT of just a few micrometers. Fabrication of a pure self supporting PZT beam without any mechanical support structure will be very difficult using these methods. We therefore use thick film screen printing technique, and utilize the advantage of being able to screen print PZT thick film with a thickness of 15 µm to 60 µm to fabricate energy harvesters with a pure PZT beam [18].

Figure 1: The bimorph MEMS energy harvester on top of a CR 2032 button battery.

In this work the piezoelectric PZT thick film used is InSensor® TF2100; the PZT film was screen printed with a thickness of around 30 µm. This technique has previously been used in fabrication of a piezoelectric accelerometer [19] and energy harvester [20]. With the use of PZT thick film, instead of PZT thin film, a mechanical support material is no longer needed in the final device, since the beam is thick and strong enough to support the proof mass. The fabricated bimorph cantilever energy harvester depicted in Figure 1 consists of two PZT layers separated by a middle electrode. Here both layers are active and thus the strain energy from both layers is...
harvested [21]. The advantage of such a structure is that all strain energy is harvested, while in conventional structures with inactive support materials the strain energy in these is not harvested and thus wasted.

FABRICATION

The PZT/PZT thick film energy harvesters are fabricated using a six mask fabrication process on a 4 inch (100 mm) silicon on insulator (SOI) wafer with a 30 µm device layer, a 1 µm buried oxide layer and a 525 µm thick carrier substrate, Figure 2(a). The backside of the SOI wafer is patterned using UV lithography and etched using a DRIE process, Figure 2(b). The buried oxide layer is here used as an etch stop layer. After an RCA clean a 1 µm thick silicon dioxide layer is grown in a furnace by thermal oxidation. This layer will serve as an etch stop layer. After an RIE etch, releasing the structure, Figure 2(c). The bottom electrode is deposited on the front side of the SOI wafer using e-beam deposition. The bottom electrode layer consists of a 50 nm titanium adhesion layer and 500 nm of platinum which also serves as a diffusion barrier in the PZT sintering process. The bottom electrode layer is patterned using UV lithography and etched in a wet etch solution, H₂O:HCl:HNO₃ (8:7:1) at elevated temperature, Figure 2(d). On top of the patterned bottom electrode a 30 µm PZT thick film is deposited using screen printing and sintered (Figure 2(e)). The middle electrode is then deposited through a prefabricated shadow mask using e-beam deposition, Figure 2(f). The shadow mask was made using a 350 µm silicon wafer, which was patterned using UV lithography and etched through in a DRIE process. Then the second layer of PZT thick film is screen printed and sintered. The top electrode layer is similarly deposited through a shadow mask using e-beam deposition, Figure 2(g). With the two PZT stacks in place the silicon device layer can now be removed. First, the oxide on the backside is etched in BHF, while the front side of the SOI wafer is protected. The backside of the SOI wafer is then etched in a RIE process until the device layer is removed and the cantilever released, Figure 2(h). The wafer is diced and the chips polarized individually. The polarization directions of the two layers are aligned opposite to each other, i.e. during polarization the top and bottom electrodes are grounded and a polarization voltage is applied to the middle electrode. The dimensions for the final energy harvester chips are shown in Table 1.

Table 1: Energy harvester dimensions.

<table>
<thead>
<tr>
<th>Frame dimensions</th>
<th>10 mm × 10 mm</th>
</tr>
</thead>
<tbody>
<tr>
<td>Medial dimensions</td>
<td>&lt; 1 mm</td>
</tr>
<tr>
<td>Cantilever width</td>
<td>5.5 mm</td>
</tr>
<tr>
<td>Cantilever length</td>
<td>1.95 mm</td>
</tr>
<tr>
<td>Total cantilever height</td>
<td>2 × 30 µm</td>
</tr>
<tr>
<td>Mass length</td>
<td>4.55 mm</td>
</tr>
</tbody>
</table>

The fabricated energy harvester was characterized both visually and electrically. The electrical characterization is done by measuring the direct and the indirect piezoelectric effect. The indirect piezoelectric effect was measured using an Agilent 4294A Precision Impedance Analyzer to excite the harvester electrically, and then measure the impedance and the phase.

Figure 3 shows one of such measurements where the bottom electrode and the top electrode are probed. The resonant frequency is fₑ = 344 Hz and the anti-resonant frequency is fₛ = 347 Hz. The impedance at resonance is the optimal resistive load Rₛ = 405 kΩ. The same measurements were done between the bottom and middle electrode and between the middle and top electrode. Even though the resonant and anti-resonant frequency stays the same, the optimal resistive load will change mainly due to the change in the layer thickness. The optimal resistive
load between bottom and middle electrode is $R_{opt} = 179 \, k\Omega$. The optimal resistive load between middle and top is $R_{opt} = 345 \, k\Omega$.

The direct piezoelectric effect is measured using the aforementioned resistive loads in a shaker setup. The harvester is actuated by a B&K Mini Shaker 4810 driven by an amplified sinusoidal signal from an Agilent 33220A function generator. A B&K Piezoelectric Accelerometer 8305 is mounted on the shaker along with the harvester. The acceleration from the shaker was measured by the accelerometer and is stated in fractions of the gravitational acceleration $g$ ($9.81 \, m \, s^{-2}$). Since the PZT layers are polarized in opposite directions, the power output of both layers was measured with the top electrode and the bottom electrode, connected by the resistive load $R_{opt} = 405 \, k\Omega$. By measuring the RMS voltage drop across the load the dissipated power can be calculated as $P_{rms} = V_{rms}^2 / R_{opt}$. Figure 4 shows the power output as a function of the excitation frequency. A maximum power of $7.35 \, \mu W$ was measured at an acceleration of $1 \, g$. The asymmetric shape of the power peak and the decrease in resonant frequency with increasing power is probably due to a non-linear response of PZT under stress; with increasing stress in PZT the material’s effective Young’s modulus and the quality factor decreases [22]. The decrease in Young’s modulus decreases the spring constant and thus also the resonant frequency.

During measurements of the power output of the top PZT layer, the top electrode and the middle electrode was connected through the resistive load of $R_{opt} = 345 \, k\Omega$, while the bottom electrode is kept in open circuit. In a similar to measurement of the power output of the bottom PZT layer the resistive load of $R_{opt} = 179 \, k\Omega$ was used, and the top electrode was kept in open circuit. Similar measurements to those shown in Figure 4 are done on the top and bottom PZT layer. The power output from these measurements at resonant frequency are collected and plotted against the input acceleration, as shown in Figure 5. Figure 5 shows that the power output is indeed higher when both layers are measured, but it is not the sum of the power outputs from the other two layers. This is caused by the factor that when more power is extracted from the harvester the more electrically damped the system becomes and thus yields lower power.

The difference in optimal resistive load and power output from the bottom and the top PZT layer is caused by mismatch in PZT film thickness and material properties, which is confirmed by inspection of the cantilever cross section using scanning electron microscopy (SEM). Figure 6 shows the cross section of the PZT/PZT cantilever, and illustrates clearly the undesirable layer mismatch. Besides the bottom PZT layer is thinner than the top layer, the grains of the bottom layer appear to be larger.

![Figure 4](image1.png)

*Figure 4: Power output as a function of the excitation frequency for different accelerations. The top electrode was connected to the bottom electrode through a resistive load, $R_{opt} = 405 \, k\Omega$, optimized for maximum power using the impedance analyzer measurements.*

![Figure 5](image2.png)

*Figure 5: The power output from the top PZT layer, the bottom PZT layer and both layers at resonant frequency as a function of the input acceleration.*

![Figure 6](image3.png)

*Figure 6: A SEM image showing a cross sectional view of the PZT/PZT bimorph cantilever. The middle electrode separates the two PZT layers.*
CONCLUSION

Screen printed PZT/PZT thick film bimorph MEMS cantilever energy harvesters were successfully fabricated and characterized. By replacing the inactive cantilever support with another layer of PZT more power is indeed harvested compared to the power harvested from each individual layer. At 1 g the power output from both layers combined is 7.35 µW, while the power output for the individual layers at same input acceleration are 5.39 µW for the bottom layer and 3.62 µW for the top layer.

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REFERENCES


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