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Preisach model of hysteresis for the Piezoelectric Actuator Drive

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Abstract — The Piezoelectric Actuator Drive (PAD) is a precise piezoelectric motor generating high-torque rotary motion, which employs piezoelectric stack actuators in a wobble-style actuation to generate rotation. The piezoelectric stacked ceramics used as the basis for motion in the motor suffer from hysteretic nonlinearities. In order to model these nonlinearities, the first-order hysteresis reversal curves of the actuators are measured and a discrete Preisach model is derived. This forms a basis that enables the study of different compensation methods. The results show matching between measured and estimated responses within 95.8%

Keywords — motor, actuator, piezoelectric, stack, multilayer, hysteresis, model, Preisach

I. INTRODUCTION

The Piezoelectric Actuator Drive (PAD) shown in Fig. 1 is a type of rotary motor that transforms the linear motion of piezoelectric stack actuators into a precise rotation [1]. The operating principle, illustrated in Fig. 2, is based on a wobble-style motor, where the motor ring is actuated into an off-center circular motion around an axially constrained shaft, thereby generating rotational motion. A micro-mechanical toothing interface is machined between the ring and shaft, as shown in Fig. 3. This enables both high positioning accuracy and output torque [2]. The actuation method, combined with the type of toothing and the inherent large stiffness of the piezoelectric stack actuators makes the PAD appropriate for open-loop control [3].

By construction, the PAD motor behaves like a fully capacitive rotary machine, due to the piezoelectric stack actuators it employs. These actuators exhibit desirable properties such as large stiffness and a reversibility of the piezoelectric effect, enabling them to act as both actuators and sensors [4]. Nevertheless, two dominating nonlinearities persist in the control of piezoelectric-based actuation, namely hysteresis and creep [5]. Of the two effects, hysteresis has a much larger impact on the positioning precision of the motor, especially under dynamic operation. The purpose of this paper is to model this effect in order to correctly predict the response of the PAD.

Modeling hysteresis in the piezoelectric motor is a necessary step in developing compensation methods to negate the effects. The most encountered method of modeling hysteresis in literature remains the Preisach-Krasnoselskii scalar model [6], which provides a straightforward method of representing the hysteretic effect. Although initially developed for the purpose of modeling magnetic hysteresis, due to its fenomenological nature, the method can be used to model any type of hysteresis [7]. This fact, combined with the generally predictable performance has led the Preisach model to be the preferred method used when hysteresis is involved.

Many modifications and improvements to the original model have been proposed, such as accounting for dynamic input changes [8], [9] or reducing the computational complexity by simplifying the parameter identification process [10]. For this paper, simplicity and real-time execution are the most important factors in determining the method used. Therefore, the classical model enhanced with Mayergoyz’s observations on determining the Everett function from the first-order inverting loops [6], [11] is chosen. The paper starts by presenting the underlying principle of the classical Preisach model in section II, followed by a short description of the experimental set-up and tests performed in section III. The model response is then compared to experimental measurements in Section IV, after which the conclusion is presented in section V.

II. THE CLASSICAL PREISACH MODEL

The idea behind the classical scalar Preisach model consists of describing the hysteretic effect through the use of an
infinite number of parallel-connected relay-type two-state discontinuous operators $\gamma_{\alpha\beta}[u(t)]$, called basic hysterons. These elements represent relay-like loops on the input-output plane with two values, $\alpha$ and $\beta$, acting as the 'on' and 'off' thresholds, with $\beta \leq \alpha$. When the input signal $u(t)$ becomes greater than $\alpha$, the output of the operator becomes 'high' and goes to 'low' state only when the input becomes less than $\beta$. In between thresholds, the previous value is retained. More explicitly, in the case of piezoelectric devices this can be written mathematically as

$$
\gamma_{\alpha\beta}[u(t)] = \begin{cases} 
0 & u(t) \leq \beta; \\
\gamma_{\alpha\beta}[u(t)] & \beta \leq u(t) \leq \alpha; \\
1 & u(t) \geq \alpha.
\end{cases}
$$

This representation of the basic hysteretic operator, represented graphically in Fig. 5, is in agreement with the property of a hysteretic system where the output depends not only on the input, but past values of the input as well. In other words, hysteresis is a nonlinearity with a non-local memory effect.

In order to be able to reproduce a wide range of hysteresis curves, the outputs of the basic hysterons are then weighted with an experimentally-determined weighting function $\mu(\alpha, \beta)$. Determining this function represents the major challenge in deriving and using the Preisach hysteresis model.

The structural block diagram of the Preisach model is shown in Fig. 4. It illustrates the parallel connection and weighting of individual elementary hysterons, whose sum produces the instantaneous output. Mathematically, this is expressed as

$$
x(t) = \int_{\alpha\geq\beta} \mu(\alpha, \beta) \gamma_{\alpha\beta}[u(t)] \, d\alpha \, d\beta
$$

where $u(t)$ is the voltage input to the piezoelectric actuator, $x(t)$ represents the output displacement, $\mu(\alpha, \beta)$ is the experimentally-obtained weighting function and $\gamma_{\alpha\beta}[u(t)]$ is the hysteron state.

A geometrical interpretation of the hysteron plane (also called the $\alpha - \beta$ plane) greatly facilitates the understanding of (2) and the Preisach model in general. In this plane, a so-called
Preisach triangle $T_0$ is defined, which represents the region of operation of the actuator, bordered by $\alpha_{\text{max}}, \beta_{\text{max}}, \alpha_{\text{min}}$ and $\beta_{\text{min}}$. Only the surface above the diagonal given by $\alpha = \beta$ has any physical meaning and therefore $T_0$ is an upper triangular surface. The elementary hysterons $\gamma_{\alpha\beta}[u(t)]$ have a direct correlation to the $\alpha - \beta$ half-plane in such a way that at any point in time $T_0$ is divided into two surfaces $S^+$ and $S^-$, representing the $(\alpha, \beta)$ pairs for which $\gamma_{\alpha\beta}[u(t)] = 1$ or $0$, respectively. This is in stark contrast to the method being applied to describe magnetic hysteresis, where in order to wipe out the magnetization, a negative input has to be applied. In other words, in piezoelectric phenomena a lack of excitation field results in a zero net output.

Thereby, for a monotonic increase of the input $u(t)$, the input-output plane shows an ascending hysteresis branch, while the $\alpha - \beta$ half-plane 'fills up' from the bottom to the horizontal line defined by $\alpha = \{\alpha_1 | \alpha_1 \leq u(t)\}$. Similarly, a monotonic decrease in input will then determine the surface to 'empty', but this process is orthogonal to the one for increasing input. Therefore the 'filled' space $T_0$ will empty starting from the right towards the vertical line defined by $\beta = \{|\beta_1 | \beta_1 \geq u(t)\}$. Thereby a stochastic input signal with several extrema will be represented as a combination of 'filled' and 'emptied' areas on the triangle, delimited by a boundary staircase layer, denoted $L$. The problem then boils down to finding the area under the obtained staircase curve. This process is visually exemplified in Fig. 6.

An important property of the Preisach model is the wipe-out feature, whereby the input history is erased when the input increases above (or decreases below) the previous points of extrema. Specifically, in Fig. 6d if the input were to increase beyond $\alpha_1$, then the pair $(\alpha_1, \beta_2)$ would be removed from the history. Visually, the corner represented by $(\alpha_1, \beta_2)$ on the half-plane would be 'engulfed' by the vertically increasing $S^+$ surface. In time domain, this coincides with $u(t)$ increasing in amplitude beyond its previous extremum. This feature is important in order to prevent excessive history growth.

Using the presented geometric interpretation, the Preisach model (2) can be rewritten as

$$x(t) = \int_{S^+} \mu(\alpha, \beta) \gamma_{\alpha\beta}[u(t)] \, d\alpha \, d\beta + \int_{S^-} \mu(\alpha, \beta) \gamma_{\alpha\beta}[u(t)] \, d\alpha \, d\beta$$

(3)

Considering that in the case of piezoelectric actuators $\gamma_{\alpha\beta}[u(t)] = 0 | u(t) | \leq \beta$ and $\gamma_{\alpha\beta}[u(t)] = 1 | u(t) | \geq \alpha$, (3) can be reduced to

$$x(t) = \int_{S^+} \mu(\alpha, \beta) \, d\alpha \, d\beta$$

(4)

The next step in completing the Preisach hysteresis model is determining the $\mu(\alpha, \beta)$ weighting function. This is done more easily by using the system's first-order reversal curves, shown in Fig. 7a, to construct the describing Everett function [12], thereby eliminating the computationally-intensive double integration. A term denoted with $x_\alpha$ is defined as the piezoelectric expansion on the limiting ascending branch of the hysteresis curve corresponding to an input value of $\alpha$ and, similarly, $x_{\alpha\beta}$ is defined as the expansion on the descending branch that starts at the previous $x_\alpha$. Hereby, the Everett function is defined as

$$X(\alpha, \beta) = \frac{1}{2}(x_\alpha - x_{\alpha\beta})$$

(5)

By choosing two arbitrary points $x_\alpha$ and $x_{\alpha\beta}$ as shown in Fig. 7b and expressing them in terms of Preisach equation, the geometric interpretation of the Everett function shows that:

$$x_\alpha = \int_{S^+ + T_1} \mu(\alpha, \beta) \, d\alpha \, d\beta$$

(6)
\[ x_{\alpha \beta} = \iiint_{S^+} \mu(\alpha, \beta) \, d\alpha \, d\beta \]  
\[ X(\alpha, \beta) = \frac{1}{2} \iiint_{T_1} \mu(\alpha, \beta) \, d\alpha \, d\beta \]  
\[ x(t) = \begin{cases} 
X(u(t), \beta_N) + \\
\sum_{i=1}^{N} (X(\alpha_i, \beta_{i-1}) - X(\alpha_i, \beta_i)) \quad \hat{u}(t) > 0; \\
X(\alpha_N, \beta_{N-1}) - X(\alpha_N, u(t)) + \\
\sum_{i=1}^{N} (X(\alpha_i, \beta_{i-1}) - X(\alpha_i, \beta_i)) \quad \hat{u}(t) < 0.
\end{cases} \]
single piezoelectric actuator from the four. The other actuators are assumed to be identical with the measured one. This assumption is reasonable, within the fabrication tolerances of the actuators themselves but it will of course impact the model accuracy.

Fig. 10a shows the receding sawtooth voltage signal applied to the stack actuator, as measured from the output of the voltage amplifier. The frequency of the signal is 0.5 Hz, it has a length of 50 periods and a linear recession slope of $-2 \text{ V per period}$. The low frequency is chosen so that the dynamic properties of the actuator do not affect its response.

The measured displacement of the actuator corresponding to the input voltage is shown in Fig. 10b. It has the same frequency and length as the input signal, but the envelope of the recession shows an exponential response with a sub-unitary exponent, instead of the linear recession seen in the voltage signal. The glitches present in the signals at around the 45 second mark are the result of the measurement system buffer overflowing and missing some samples. A new set of measurements needs to be carried out with special attention to acquisition buffer size and amount of data.

Nevertheless, the response estimated by the developed model is in good agreement with the measured response. Fig. 11 shows a comparison between the real and modeled displacements for an applied voltage signal swinging between $31 \text{ V}-173 \text{ V}-33 \text{ V}-171 \text{ V}$. This specific input voltage swing was chosen at random out of the ranges for which measurements exist, so as to be able to evaluate the model response. The relative error plot in Fig. 11c illustrates that the model response matches the measurements within 95.8%. Therefore the model correctly estimates the actuator’s exponential displacement response as a direct result of the triangular input excitation.

VI. CONCLUSIONS

This paper models the effects of hysteresis in the piezoelectric stack actuators that make up the Piezoelectric Actuator Drive motor. A discrete Preisach model was chosen due to its popularity as a hysteresis modeling tool, relative simplicity and ease of implementation. The model was built based on the measurement of 50 hysteresis reversal curves, from which the corresponding Everett functions were extracted. Validation against a randomly-chosen input variation was done. Finally, the developed and implemented discrete Preisach model allows estimating the displacement of the stacks within the motor with a relative accuracy of 95.8% and serves as a good starting point for studying and developing different compensation algorithms that are able to eliminate hysteresis.

Future work will focus both on deriving an inverse model for compensation, as well as studying improvements to the developed model. Increasing the number of reversal curves measured to obtain higher accuracy, as well as investigating the effects of higher frequency dynamics on the model are being considered.

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Fig. 10: Measurement of the first order reversal curves used to extract the model parameters. (a) shows the receding sawtooth signal applied to the piezoelectric stack actuator, measured from the output of the amplifier. (b) is the measured displacement of the piezoelectric actuator. (c) shows the actual reversal curves obtained.

Fig. 11: Result of the model compared with the actual measured output for an input voltage pattern swinging between 31 V-173 V-33 V-171 V. (a) shows the comparison in time, (b) shows the matching hysteresis curves and (c) contains the relative error plot.